

Barrel Couplings



RINGFEDER[®] TNK



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RINGFEDER[®] Barrel Couplings

Introduction

The barrel couplings complement the portfolio of the well-proven RINGFEDER[®] couplings for crane and hoisting gear applications. The comprehensive range of RINGFEDER[®] drive couplings that connect the driving motor with the gearbox and which are generally equipped with brake discs, or brake drums, where required, have been offered for decades as a package solution by brake manufacturers and distributed under their own name.

The optimized barrel coupling series for connecting the gearbox output shaft with the rope drum extends the RINGFEDER[®] portfolio of drive components and increases the interest of manufacturers and operators in selecting RINGFEDER[®] as their direct and reliable supplier for crane components.

The particular feature of the barrel coupling, i.e. to transmit the torque via barrel-shaped hardened bodies being embedded in the spaces formed by the semi-circular toothings of the hub and the sleeve, ensures the safe transmission of radial forces while simultaneously compensating angular misalignments of the connected units. The barrel coupling thus represents an articulated joint that turns a statically indeterminate system to a statically determinate one and, as a result, compensates operation-related deformations and prevents constraining forces.

The wide range of RINGFEDER® drive couplings comprises the elastomeric, shock absorbing and fail-safe claw couplings of type RINGFEDER® TNS, the RINGFEDER® TNB couplings for very high torques, the gear couplings of type RINGFEDER® TNZ, to accommodate larger shaft misalignments and the maintenance-free steel disc couplings RINGFEDER® TND which are preferably equipped with brake discs, and optionally with brake drums.



Configuration of a double rope drum drive in a hoisting gear





Statically indeterminate because of three-point bearing. Misalignment errors cause considerable undesired reaction forces.



Statically determinate situation produced by the barrel coupling (joint). Discrepancies from the alignment will be balanced out.



RINGFEDER® TNK TKVO

Performance and Application Optimized Type, Construction acc. to Operation Sheet of the German Steel Iron Industry SEB 666212

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The barrels in the coupling typically dispose of an axial clearance so that axial movements between the rope drum and the gearbox are compensated. In such a configuration, the joint acts like a non-locating bearing. The inner cover integrated in the housing allows the hub diameter and thus also the bore diameter to be increased and fixes the barrel roller axially to the drum. To the outside and thus in the direction of the gear unit, axial fixation is provided by a retaining ring, which is also supported by an additional thrust ring. This construction complies with the specifications of the operation sheet of the German Steel Iron Industry SEB 666212. In some applications, the barrel coupling has to take the role of a fixed bearing what can be realized by design modifications. The use of high-strength materials allows a considerable increase of the transmission capability without requiring any change of the design and overall dimensions. As a result, a smaller coupling size can often be selected. The lower weight and the resulting lower acceleration forces contribute significantly to the energy efficiency of the crane.



Figure 1: Configuration and components of a barrel coupling

The torque that is induced via the gearbox output shaft is transfered over the hub and the barrel rollers into the housing. The housing transmits the torque into the drum over the bolting and two carrier faces provided on the rope drum.

The lip seal on the inside and the lip seal of the outer cover prevent foreign bodies from ingressing and lubricant from leaking out.

An indicator that is attached to the external cover (No. 5, Figure 1) allows to check the wear and the axial position (No. 10, Figure 1) of the coupling housing relative to the coupling hub. If the barrel coupling has to be dismounted, assembly markings ensure easy re-assembly at a later point of time.



Non-locating bearing design with axial securing of the barrel bodies, internally by integration in the hub, externally with retaining ring and additional thrust ring





- 1. Hub
- 2. Sleeve
- 3. Barrel roller
- 4. External cover
- 6. Sealing ring
- 7. Retaining ring
- 15. Housing screws



Coupling Size Selection

Considering the group classification according to FEM or DIN, the coupling size is determined on basis of:

1. the torque to be transmitted

- 2. the applied radial load
- 3. verification of the geometric dimension

1. Selection on basis of the torque to be transmitted

For the coupling size selection applies:



It can be calculated on basis of:

a) maximum motor power or installed powerb) required motor power

a) Calculating the torque on basis of the maximum motor power Pi

In this approach, the power reserve of the motor is included in the calculation of the torque:

| Τĸ | = | 9550 * Pi / n * k ₁ | [Nm] |
|----------------|---|----------------------------------|-------|
| Τĸ | = | Coupling torque at the rope drum | [Nm] |
| Pi | = | Installed motor power | [kW] |
| n | = | Rotary speed of the rope drum | [rpm] |
| k ₁ | = | Service factor | [-] |

Service factor k₁ to the group classification acc. to (*)

| | Class | | |
|------------------|------------------------------|----------------------------------|--------------------------|
| DIN 15020 (1974) | FEM (1970) | FEM 1.001 (1998) BS466 (1984) | Factor k ₁ |
| 1 Bm | IB | M1, M2, M3 | 1,12 |
| 1 Am | IA | M4 | 1,25 |
| 2 m | Ш | M5 | 1,4 |
| 3 m | Ш | M6 | 1,6 |
| 4 m | IV | M7 | 1,8 |
| 5 m | V | M8 | 2 |
| | L4-T8-M8; L3-T9-M8; L4-T9-M8 | | 2,2 |

(*) Service factor according to standardized calculation method, version (year)

b) Calculating the torque on basis of the required power P_N

In this approach, the torque required to lift the load is calculated taking into account the system-related additional forces:

| Ρ _Ν | = | or | [kW] |
|----------------------------------|---|---|-------------|
| Τκ | = | | [Nm] |
| Τκ | = | | [Nm] |
| P _N F _R | = | Required power Entire tackle at the drum, including the lifting gear paying regard to the efficiencies and drum bearings | [kW] [N] |

(see 2. selection on basis of the radial load)

| v _T | = | Rope velocity at the drum | [m/min] |
|----------------|---|--|---------|
| D | = | Effective winding diameter at the drum | [m] |

a) Determination of the radial load acting on the

acting on the barrel coupling

2. Selection on basis of the radial load

barrel coupling

The radial force F_S acting on the barrel coupling is composed of the pull on account of the working load and weight of the lifting device and of the impact of the rope sheave and the efficiency of the bearings.

| Fs | = (Q + G) / (ir * η) | [N] |
|----|---|--------|
| Q | = Max. force of the working load | [N] |
| G | = Weight force of the lifting device and the rope (m *g | g) [N] |
| m | = Mass | [kg] |
| g | = 9.81 (gravity acceleration) | [m/s²] |
| η | Efficiency of the support bearing and lifting | [-] |
| | device bearing | |
| | | |

The transmission ratio **ir** is determined as a function of rope fastening, number of pulley and tackles:

ir = Transmission ratio

= Total number of rope lines in the lifting device Number of rope lines to the drum [-]

| Efficiency η | | | | | | | |
|----------------------------|------|------|------|------|------|------|------|
| ir | 2 | 3 | 4 | 5 | 6 | 7 | 8 |
| η with plain bearings | 0,92 | 0,9 | 0,88 | 0,86 | 0,84 | 0,83 | 0,81 |
| η with ball bearings | 0,97 | 0,96 | 0,95 | 0,94 | 0,93 | 0,92 | 0,91 |





The transmission ratio and the used bearings in the lifting device determine the efficiency $\boldsymbol{\eta}$

b) Calculation of the radial load acting on the barrel coupling

In this approach, the impact of an inclined rope guide by the tackles and pulleys and the drum diameter is neglected because these effects, calculated by the equation below, reduce the radial force F_T acting on the barrel coupling.

For systems with two ropes at the drum, see fig. 2 & 3

$$F_{T} = F_{S} / 2 + W / 2$$

For systems with only one rope at the drum, see fig. 4 & 5

 $F_T = F_S (1 - b / L) + W / 2$

| W | = Weight of the drum including rope and the | |
|---|---|-----|
| | connected parts of the barrel coupling | [N] |
| b | = Minimal distance between rope and joint | |
| | of the barrel coupling | [m] |
| L | = Distance between the bearing locations | [m] |
| | of the rope drum | |
| | | |



According to the design rules of FEM 1.001, version 1998 and BS466, version 1984, the safety factor k_2 has to be considered for the radial load:

| Safety factor k ₂ | | | | |
|------------------------------|------|-----|------|-----|
| Load spectrum | L1 | L2 | L3 | L4 |
| k ₂ | 1,05 | 1,1 | 1,15 | 1,2 |

The value F_R which is to be calculated for the selected coupling must be lower than the permissible radial load F_Rmax indicated in the tables.

[Nm]

 $F_R = F_T \cdot k_2 < F_{Rmax}$

c) Size optimization on account of permissible correction of the calculation

The loads due to torque and radial force correlate, so that in case one of the maximum values is not fully used up, the other value can be corrected. This may in some cases allow the use of a smaller coupling size.

Case 1: Radial load correction -> F_C [Nm]

The torque capability of the selected size is not yet fully used and the calculated radial force FR is above the permissible value. It is:

$$F_C = F_{Rmax} + (T_{max} - T_K) \cdot C > F_R \text{ and } F_C < 1.5 \cdot F_{Rmax}$$

Case 2: Torque correction -> T_C [Nm]

The calculated torque is just above the permissible torque of the coupling, however, the permissible radial load FRmax is not yet fully used up. It is:

$$T_C = T_{max} + (F_{Rmax} - F_T) / (C \cdot k_1)$$
 and $T_C < 1.08 \cdot T_{max}$

| C | orrection factor | C for T _{Kmax} /F _{Rmax} | |
|---------------|------------------|--|-----|
| Coupling size | с | Coupling size | с |
| 25 | 14,8 | 1000 | 4,4 |
| 50 | 13,7 | 1500 | 3,7 |
| 75 | 11,4 | 2100 | 3,6 |
| 100 | 10,8 | 2600 | 3,3 |
| 130 | 9,0 | 3400 | 3,3 |
| 160 | 8,7 | 4200 | 2,9 |
| 200 | 7,4 | 6200 | 2,6 |
| 300 | 7,2 | 8200 | 2,4 |
| 400 | 6,1 | 9200 | 2,2 |
| 500 | 5,3 | 10200 | 1,9 |
| 600 | 4,8 | | |

3. Checking the connection geometry

As a standard, the hubs of the barrel couplings are equipped with 2 keyways offset by 120° according to DIN 6885-1. The position of the keyways is always specified in viewing direction of the rope

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drum. Other hub-shaft connections can also be provided. The transmission capability of the hub-shaft connection has to be checked for each type of connection. If an interference fit is used as a hub-shaft connection, the influence of the interference on the functionally required clearance of the barrel coupling has to be checked by us.

The installation of a hub with interference fit is often done in warm condition of the hub. This requires the prior removal of the barrel coupling. While assembling the individual components, the position marking must always be observed. This marking can be found on a tooth opposite to the wear marking.



The position of the keyways is always defined in viewing direction of the rope drum.

The flange is included in the rope drum through the face S (h9/F8) to secure the torque transmission capacity, as is standard for rope drums. The connecting screws must have at least strength category 10.9.

The position of the keyways is always defined in viewing direction of the rope drum.



Barrel Couplings RINGFEDER® TNK TKVO



Performance and Application Optimized Type

The type **RINGFEDER® TNK TKVO** is a torsionally stiff barrel coupling with hub and housing made of high-strength material. The inner ring integrated in the housing enables quick and easy assembly with an extremely compact coupling design. The type incorporates construction according to the operation sheet of the German Steel Iron Industry SEB 666212 for rope drum hinge joints.



Characteristics

- Torsionally rigid, compensate for angular and axial shaft misalignment
- Torque transmission by steel rollers
- With standardized connection to rope drums in crane lifting gear
- Reduced assembly effort due to integrated inner ring as well as simplified insertion of the barrel coupling
- With wear indicator for easier condition monitoring
- Bore diameter d₁ up to 440 mm
- Outer diameter D up to 850 mm
- Transmissible torque T_{Kmax} up to 815,000 Nm
- Maximum radial load Frad up to 490,000 N







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Barrel Couplings **RINGFEDER® TNK**

Tables & Values

 H_2

For the correct selection of a suitable coupling size, it is essential to observe the equations and recommendations in the chapter "Coupling Size Selection".

 H_5



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Barrel Couplings RINGFEDER® TNK TKVO

Performance and Application Optimized Type





| Identifier | SEB 666212 | Size | T _{Kmax} | F _{rad} | d _{1kmin} | d _{1kmax} | D | D ₂ | L | L _{min} | D1 | D ₆ | D ₈ | H1 | R | H ₂ | Fĸ | H ₅ | YL |
|------------|---------------|------|-------------------|------------------|--------------------|--------------------|-----|----------------|-----|------------------|-----|----------------|----------------|------|-----|----------------|----|----------------|------|
| | | | Nm | N | mm | mm | mm | mm | mm | mm | mm | mm | mm | mm | mm | mm | mm | mm | mm |
| WG3125 | (*) | 25 | 6500 | 17650 | 20 | 70 | 250 | 110 | 95 | 85 | 95 | 159 | 160 | 16 | 2,5 | 42 | 12 | 31 | 44,0 |
| WG3128 | (*) | 50 | 8400 | 20000 | 20 | 80 | 280 | 127 | 100 | 85 | 110 | 179 | 180 | 16 | 2,5 | 42 | 12 | 33 | 45,0 |
| WG3132 | (*) | 75 | 10500 | 21500 | 20 | 90 | 320 | 142 | 110 | 95 | 125 | 199 | 200 | 17 | 2,5 | 45 | 15 | 32 | 46,0 |
| WG3134 | (*) | 100 | 16000 | 28000 | 20 | 100 | 340 | 155 | 125 | 95 | 140 | 219 | 220 | 19 | 2,5 | 45 | 15 | 34 | 47,0 |
| WG3136 | (*) | 130 | 21500 | 37000 | 47 | 115 | 360 | 175 | 130 | 95 | 160 | 239 | 240 | 19 | 2,5 | 45 | 15 | 36 | 48,0 |
| WG3138 | (*) | 160 | 27000 | 42500 | 47 | 130 | 380 | 194 | 145 | 95 | 180 | 259 | 260 | 21 | 2,5 | 45 | 15 | 35 | 48,0 |
| WG3140 | SG130 | 200 | 31500 | 48000 | 47 | 145 | 400 | 213 | 170 | 95 | 200 | 279 | 280 | 21 | 2,5 | 45 | 15 | 37 | 48,0 |
| WG3142 | (*) | 300 | 39000 | 53000 | 47 | 160 | 420 | 234 | 175 | 95 | 220 | 309 | 310 | 25 | 2,5 | 45 | 15 | 40 | 50,0 |
| WG3145 | SG140 | 400 | 53500 | 75000 | 47 | 190 | 450 | 274 | 185 | 120 | 260 | 339 | 340 | 21 | 2,5 | 60 | 20 | 39 | 60,5 |
| WG3151 | (*) | 500 | 91000 | 118000 | 77 | 215 | 510 | 314 | 220 | 125 | 295 | 399 | 400 | 29 | 2,5 | 60 | 20 | 49 | 64,5 |
| WG3155 | SG185 | 600 | 127000 | 132000 | 77 | 225 | 550 | 329 | 240 | 125 | 310 | 419 | 420 | 29 | 2,5 | 60 | 20 | 49 | 64,5 |
| WG3158 | SG200 | 1000 | 180000 | 145000 | 102 | 255 | 580 | 368 | 260 | 130 | 350 | 449 | 450 | 29,5 | 2,5 | 60 | 20 | 49,5 | 65,0 |
| WG3165 | SG240 | 1500 | 241000 | 184000 | 102 | 305 | 650 | 431 | 315 | 140 | 415 | 529 | 530 | 31,5 | 2,5 | 65 | 25 | 51,5 | 68,0 |
| WG3166 | (*) | 2100 | 360000 | 283000 | 102 | 315 | 665 | 458 | 330 | 145 | 430 | 544 | 545 | 43 | 4,0 | 65 | 25 | 63 | 74,0 |
| WG3168 | SG270 | 2600 | 425000 | 330000 | 102 | 325 | 680 | 470 | 350 | 145 | 445 | 559 | 560 | 43 | 4,0 | 65 | 25 | 63 | 74,0 |
| WG3171 | SG315 | 3400 | 529000 | 366000 | 178 | 350 | 710 | 502 | 380 | 165 | 475 | 599 | 600 | 38 | 4,0 | 81 | 35 | 63 | 86,0 |
| WG3178 | SG355 | 4200 | 660000 | 420000 | 208 | 395 | 780 | 566 | 410 | 165 | 535 | 669 | 670 | 40 | 4,0 | 81 | 35 | 66 | 87,5 |
| WG3185 | SG400 | 6200 | 815000 | 490000 | 238 | 440 | 850 | 630 | 450 | 165 | 600 | 729 | 730 | 42 | 4,0 | 81 | 35 | 65 | 87,5 |

*Construction and design acc. to operation sheet of German Steel Iron Industry SEB 666212

To continue see next page



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Barrel Couplings RINGFEDER® TNK TKVO

| Identifier | SEB 666212 | Size | с | n _{b7} | d7 | S (h9/F8) | SD | GG | L _{DD} | Xa | J _{sb} | Gw _{sb} |
|------------|---------------|------|-----|-----------------|----|-----------|----|------|-----------------|------|-----------------------------------|------------------|
| | | | mm | | mm | mm | mm | inch | mm | mm | 10 ⁻³ kgm ² | kg |
| WG3125 | (*) | 25 | 220 | 10 | 15 | 220 | 50 | G1/8 | 5 | +/-3 | 60 | 13 |
| WG3128 | (*) | 50 | 250 | 10 | 15 | 250 | 50 | G1/8 | 5 | +/-3 | 90 | 17 |
| WG3132 | (*) | 75 | 280 | 10 | 19 | 280 | 60 | G1/8 | 5 | +/-4 | 170 | 24 |
| WG3134 | (*) | 100 | 300 | 10 | 19 | 300 | 60 | G1/8 | 5 | +/-4 | 240 | 32 |
| WG3136 | (*) | 130 | 320 | 10 | 19 | 320 | 60 | G1/8 | 5 | +/-4 | 330 | 38 |
| WG3138 | (*) | 160 | 340 | 10 | 19 | 340 | 60 | G1/8 | 5 | +/-4 | 450 | 48 |
| WG3140 | SG130 | 200 | 360 | 10 | 19 | 360 | 60 | G1/8 | 5 | +/-4 | 650 | 64 |
| WG3142 | (*) | 300 | 380 | 10 | 19 | 380 | 60 | G1/8 | 5 | +/-4 | 910 | 79 |
| WG3145 | SG140 | 400 | 400 | 10 | 24 | 400 | 70 | G1/4 | 9 | +/-4 | 1520 | 108 |
| WG3151 | (*) | 500 | 460 | 10 | 24 | 460 | 70 | G1/4 | 7 | +/-6 | 3090 | 163 |
| WG3155 | SG185 | 600 | 500 | 10 | 24 | 500 | 70 | G1/4 | 7 | +/-6 | 4060 | 195 |
| WG3158 | SG200 | 1000 | 530 | 14 | 24 | 530 | 70 | G1/4 | 7 | +/-6 | 5880 | 244 |
| WG3165 | SG240 | 1500 | 600 | 14 | 24 | 580 | 80 | G1/4 | 7 | +/-6 | 12630 | 404 |
| WG3166 | (*) | 2100 | 615 | 26 | 24 | 590 | 90 | G1/4 | 6 | +/-6 | 15670 | 467 |
| WG3168 | SG270 | 2600 | 630 | 26 | 24 | 600 | 90 | G1/4 | 6 | +/-6 | 18150 | 520 |
| WG3171 | SG315 | 3400 | 660 | 26 | 28 | 640 | 90 | G1/4 | 10 | +/-8 | 25460 | 598 |
| WG3178 | SG355 | 4200 | 730 | 26 | 28 | 700 | 90 | G1/4 | 10 | +/-8 | 42020 | 795 |
| WG3185 | SG400 | 6200 | 800 | 26 | 28 | 760 | 90 | G1/4 | 10 | +/-8 | 67270 | 1049 |

*Construction and design acc. to operation sheet of German Steel Iron Industry SEB 666212

Explanations

| T _{Kmax} | = | Max. transmissible torque of the coupling | D ₁ | = Outer diameter hub | n _{b7} | Quantity of bore d₇ |
|--------------------|------------|---|----------------|--------------------------------------|------------------|--|
| F _{rad} | = | Admissible force radial | D ₆ | = Outer diameter cover | d ₇ | Bore diameter Flange |
| d _{1kmin} | - | Min. bore diameter d_1 with keyway acc. | D ₈ | = Centering diameter | S (h9/F8) | Distance of the flattening |
| | | to DIN 6885-1 | H ₁ | = Length | SD | = Disassembly Space |
| d _{1kmax} | x = | Max. bore diameter d ₁ with keyway acc. to DIN 6885-1 | R | = Radius | G _G | = Whitworth thread |
| D | _ | Outer diameter flange | H_2 | = Distance | L _{DD} | Distance dimension |
| | | Outer diameter hub | Fκ | Flange thickness | Xa | = Axial gap max. |
| D ₂ | | | H ₅ | = Distance | J _{sb} | Moment of inertia at smallest |
| L | = | Total length | YL | = Distance | | bore diameter |
| L _{min} | = | Minimum length | с | = Pitch circle diameter | Gw _{sb} | = Weight at smallest bore diameter |

Ordering example

| Identifier | Size | d _{1k} |
|------------|------|-----------------|
| WG3140 | 200 | 80 |

Technical Information

Without further specifications, we deliver as standard: Bore tolerance H7; Keyway acc. to DIN 6885-1; Keyway width tolerance P9.

Further information on **RINGFEDER® TNK TKVO** on **www.ringfeder.com**

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